



**Dynamics and
Real-Time
Simulation
(DARTS)
Laboratory**

Spatial Operator Algebra (SOA)

2. Single Rigid-Body Dynamics

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SOA Foundations Track Topics (serial-chain rigid body systems)



1. **Spatial (6D) notation** – spatial velocities, forces, inertias; spatial cross-product, rigid body transformations & properties; parallel axis theorem
2. **Single rigid body dynamics** – equations of motion about arbitrary frame using spatial notation
3. **Serial-chain kinematics** – minimal coordinate formulation, hinges, velocity recursions, Jacobians; first spatial operators; $O(N)$ scatter and gather recursions
4. **Serial-chain dynamics** – equations of motion using spatial operators; Newton–Euler mass matrix factorization; $O(N)$ inverse dynamics
5. **Mass matrix** - composite rigid body inertia; forward Lyapunov equation; mass matrix decomposition; mass matrix computation; alternative inverse dynamics
6. **Articulated body inertia** - Concept and definition; Riccati equation; alternative force decompositions
7. **Mass matrix factorization and inversion** – spatial operator identities; Innovations factorization of the mass matrix; Inversion of the mass matrix
8. **Recursive forward dynamics** – $O(N)$ recursive forward dynamics algorithm; including gravity and external forces; inter-body forces identity

6D Spatial Notation Recap



Spatial notation offers concise & consistent transformation expressions for arbitrary non-CM points

Spatial velocities ${}^C \mathcal{V}(A, C) = \underbrace{\phi^*(B, C)}_{\text{rigid body transformation matrix}} {}^B \mathcal{V}(A, B)$

Spatial forces ${}^B \mathbf{f}(B) = \underbrace{\phi(B, C)} {}^C \mathbf{f}(C)$

Spatial inertia $M(x) = \underbrace{\phi(x, y)} M(y) \underbrace{\phi^*(x, y)}$

Kinetic energy $\mathcal{K}_e = \frac{1}{2} \mathcal{V}^*(x) M(x) \mathcal{V}(x) = \frac{1}{2} \mathcal{V}^*(y) M(y) \mathcal{V}(y)$

Spatial momentum $\mathbf{h}(x) = \underbrace{\phi(x, y)} \mathbf{h}(y)$

Spatial Notation Benefits



- Reduces number of equations by half
- Reduces number of terms in each equation by over half
- Reduces types of terms needed – the $\phi(x, y)$ *rigid body transformation matrix* does much of the work across the board
- Equations apply generally, not just CM
- See consistent patterns (repetition, duality) across the different transformations
- There are useful properties involving spatial cross product and $\phi(x, y)$



Single Rigid Body Dynamics

Outline



- Dynamical systems
- Choosing coordinates
- Single rigid body dynamics



Dynamical Systems for Multibody

Dynamical systems



General form of a continuous and smooth dynamical system:

$$\begin{array}{l} \textit{state} \\ \textit{derivative} \\ \downarrow \\ \dot{\mathbf{x}}(t) = \mathbf{g}(\mathbf{x}(t), \mathbf{u}(t)) \quad \textit{state derivative equation} \\ \mathbf{y}(t) = \mathbf{h}(\mathbf{x}(t), \mathbf{u}(t)) \quad \textit{output equation} \\ \uparrow \\ \textit{output} \end{array}$$

state

input

Multibody dynamical systems



- Dynamical systems
 - **equations of motion** define the state derivative equation
 - **State x** : coordinates + velocities
 - **Inputs u** : external forces, gravity
 - **Outputs y** : are poses, velocities, loads etc
- Need to define the multibody coordinates
 - May be abstracted from physical space – but could be the same

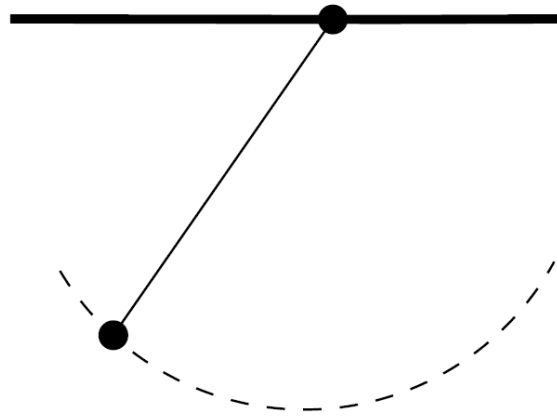


Generalized Coordinates

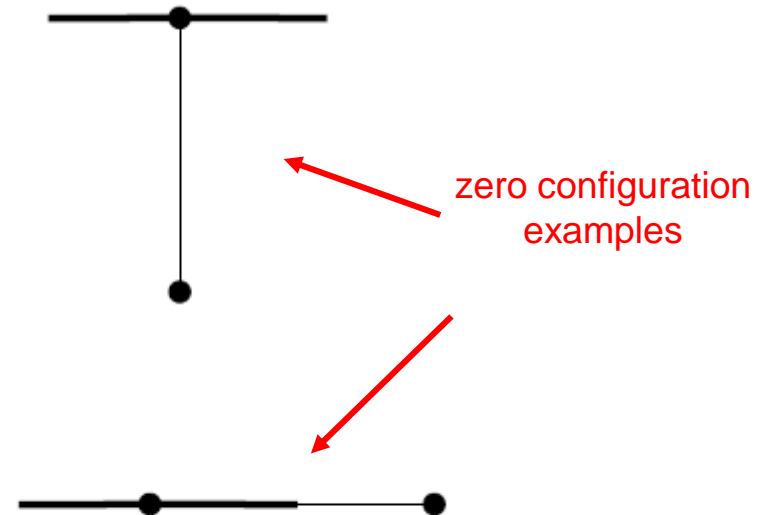


Multibody reference/zero-configuration

- The multibody zero-configuration defines the configuration where the state is zero
- Does not have to be a physically meaningful configuration
- Defines a reference configuration



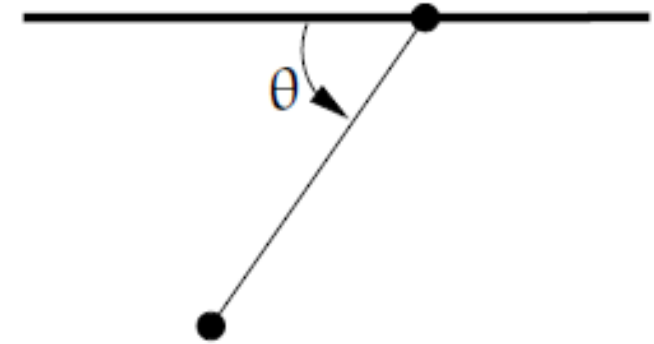
single link pendulum





Generalized coordinates θ

- **Generalized coordinates** θ specify the configuration of the system
- **Generalized velocities** are often (but not always) just the generalized coordinate rates $\dot{\theta}$
- The multibody dynamical **state** consists of generalized coordinates and generalized velocities



single link pendulum

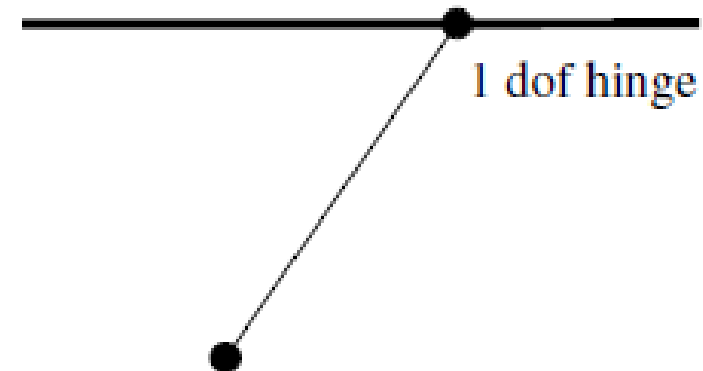


Hinges and Constraints



Motion coordinates

- Single link pendulum example
- Permissible motion can be described as
 - **Explicit:** a 1 degree of freedom (dof) **hinge**
 - **Implicit:** or equivalently as free 6 dof body, with 5 motion **constraints**

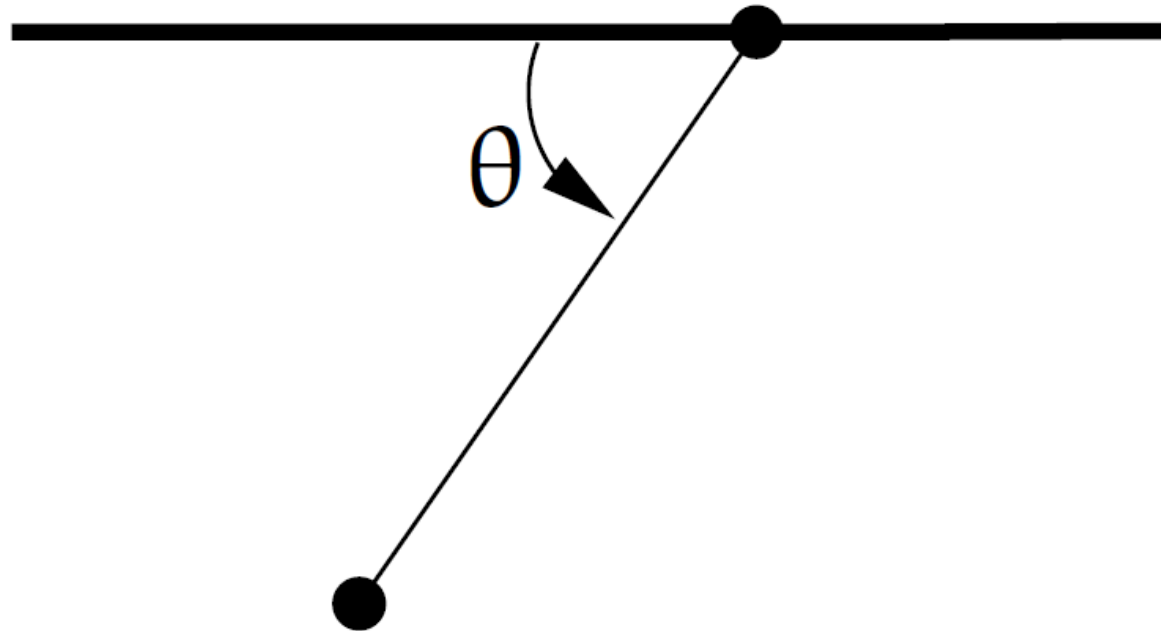


single link pendulum

Hinges minimal coordinates approach



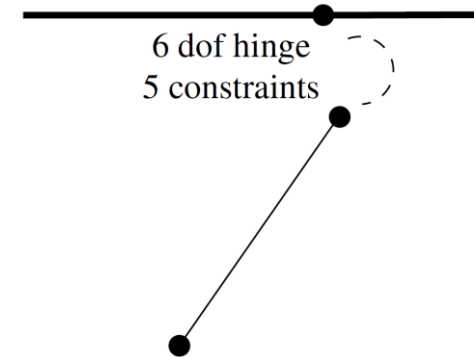
Explicit: a 1 degree of freedom (dof) hinge



Constraints approach



- **Implicit:** or equivalently as free 6 dof body, with 5 motion **constraints**

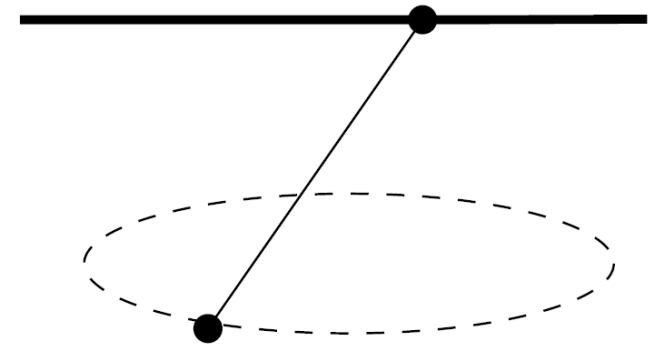


- **Alternative constraints approach: *Natural coordinates:*** more redundant coordinates using 2 points and a unit vector on the rigid body (9 dof, 8 motion constraints)
 - Avoids use of rotational coordinates

Multiple dof hinges



- We adopt the hinge **minimal coordinate approach**, and try to minimize use of constraints
- A hinge can have more than 1 dof



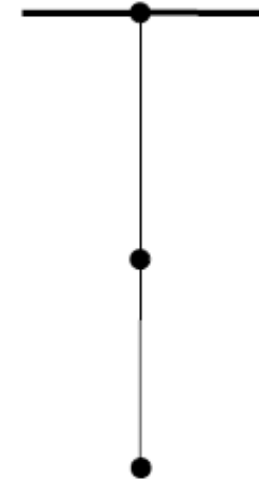


Examples of Choosing Minimal Coordinates

Example: 2 link pendulum



2 hinges and 2 dofs – overall motion is defined by the individual motion of the pair of hinges



*zero
configuration*

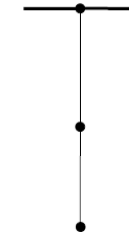
Generalized coordinates – 2 link pendulum



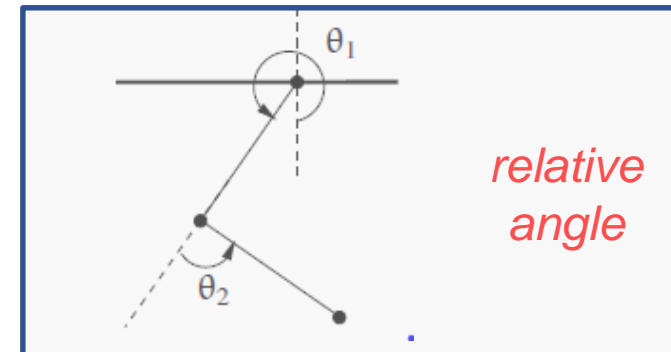
- What are the options for minimal coordinates?
- relative angle generalized coordinate

*Choice of coordinates is **not unique**
Can easily go between these
coordinate choices*

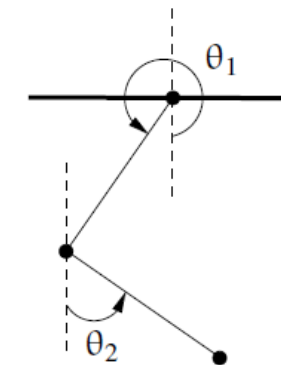
- alternative absolute angle coordinates (deviation from the vertical) which is equally valid



*zero
configuration*



*relative
angle*



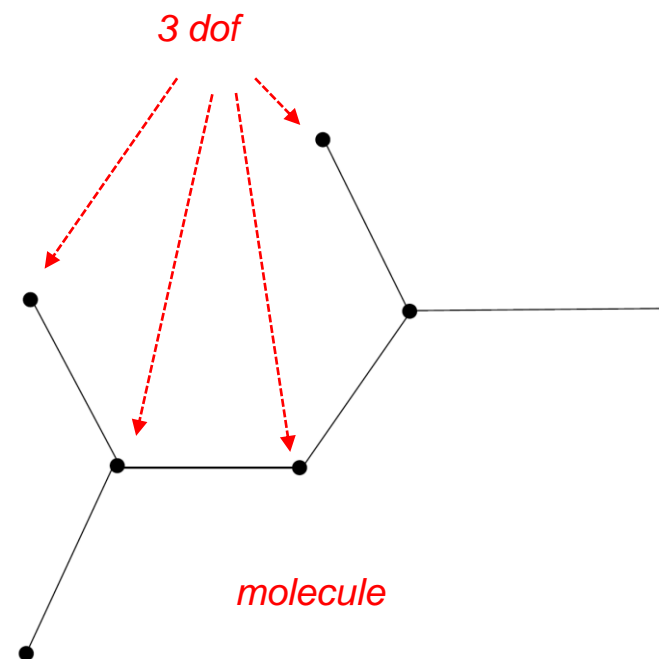
*absolute
angle*

We will use relative coordinates



Example: Molecular dynamics models

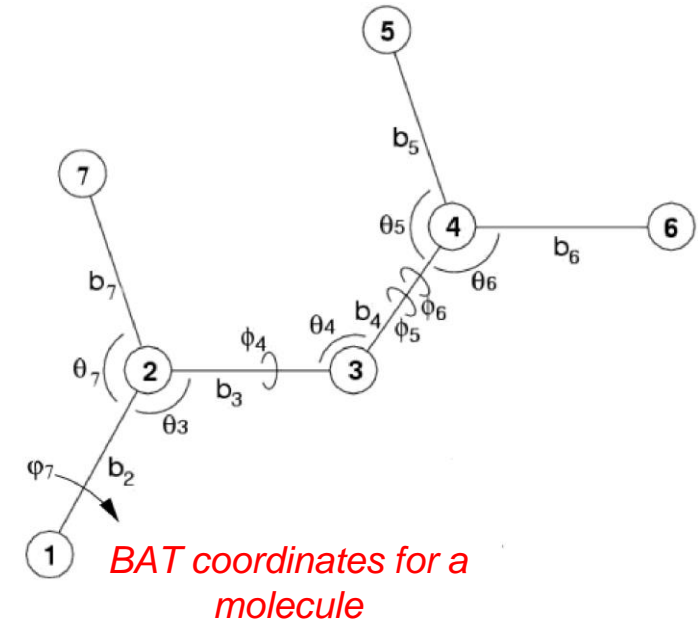
- Collection of point mass atoms, motivates **$3n$ dof** Cartesian coordinates – like for independent gas molecules
- However, with bonds, not all motion possible – bonds are stiff with little stretching and high frequency
- Common strategy is to freeze and eliminate the bond stretching dof to generate reduced order model and enable large time steps
 - This requires imposing constraints on the Cartesian coordinates





BAT coordinates for molecules

- Use alternative **bond/angle/torsion (BAT)** “internal” coordinates
- Same $3n$ number of dofs
- Coupled coordinates
- However these more naturally reflect the potentials and motion of a molecule – bond angle changes, torsional dofs
- Easy to eliminate stretching dofs by sampling ignoring these coordinates – no constraints required!
- In fact when doing entropy analysis, Cartesian motion are converted into the more appropriate BAT coordinates
 - Can avoid this by working with BAT coordinates in first place



Choosing generalized coordinates



- The choice of coordinates is not unique
- Should be based on modeling needs
- Imposes requirements on algorithms



Generalized velocities



Generalized velocities

- Generalized velocities, denoted β parameterize the velocity motion space
- A common choice is to use the generalized coordinate rates $\dot{\theta}$ as generalized velocities

$$\beta = \dot{\theta}$$

- This is fine in many cases, but does not cover all situations
 - Lets look at holonomic and non-holonomic cases



Holonomic case

- Have a function of coordinates that describes the permissible motion

$$\mathfrak{d}(\theta, t) = \mathbf{0}$$

- May be configuration and time dependent
- For **pin** hinge: $dx = dy = dz = \text{eul}(1) = \text{eul}(2) = 0$

- Use gradient $G_c(\theta, t) \triangleq \nabla_{\theta} \mathfrak{d}(\theta, t)$ (velocity constraint matrix) to obtain the velocity relationship

$$\dot{\mathfrak{d}}(\theta, t) = G_c(\theta, t) \dot{\theta} - \mathfrak{u}(t) = \mathbf{0}$$

- Check **rank r** of gradient matrix, **dofs is (6-r)**.
- Number of coordinate and velocity dofs is the same
- Orthogonal complement of gradient matrix specifies permitted relative spatial velocities across hinge



Non-holonomic case

- Start with similar velocity constraint equation

$$G_c(\theta, t)\dot{\theta} - \mathcal{U}(t) = \mathbf{0}$$

- However, in non-holonomic case, the velocity constraint matrix may **not be a gradient**
- *Fewer velocity dofs can change configuration over larger dimensional coordinate space!*
- **Examples**
 - **Ball rolling on the ground:** only 3 velocity dofs (all rotational), but can change 5 coordinates - all but z value
 - **Car:** only 2 velocity dofs, but can parallel park, i.e. change 3 coordinates – x, y and heading

Non-holonomic case (contd)



- Velocity constraint equation

$$G_c(\theta, t)\dot{\theta} - \mathcal{U}(t) = \mathbf{0}$$

- In non-holonomic case, the velocity constraint matrix may **not be a gradient**
- Check rank r of gradient matrix, **dofs is (6-r)**.
- Coordinate and velocity dofs may **not** be the same
- Orthogonal complement of the gradient matrix specifies permitted hinge motion (i.e. relative hinge spatial velocities)

Degrees of freedom for a hinge usually refer to the velocity space dofs

Quasi-velocities



- Take the case of a tumbling rigid body – has position and attitude dofs – 3 each
- With Euler angle rates, etc for generalized velocities, the attitude dynamics are complicated
- Dynamics much simpler using angular velocities

$$\mathcal{J} \dot{\omega} + \tilde{\omega} \mathcal{J} \omega = f$$

- So why not use angular velocities as generalized velocities!
- Angular velocities are however **not integrable** (i.e. not time derivatives). Not a problem.
- Such non-integrable generalized velocity coordinates are called **quasi-velocities**

Generalized forces \mathcal{T}



- For a rigid body power is given by

$$\text{power} = \mathbf{f}^* \cdot \mathcal{V}$$

- Say $\mathcal{V} = A\beta$, thus

$$\text{power} = \mathbf{f}^* \cdot A\beta = \mathcal{T}^* \beta, \text{ with } \mathcal{T} = A^* \mathbf{f}$$

- We have transformed power relationship from physical to generalized coordinates domain, and used it to define the **generalized force** \mathcal{T} .
- Given a choice for generalized velocities β , the power relationship automatically defines what \mathcal{T} should be

Example: Transforming generalized velocities



- Lets say we have a different choice for generalized velocities

$$\beta_1 = A(\theta)\beta$$

- Then by power relationship

$$\text{power} = \mathcal{T}^* \cdot \beta = \mathcal{T}^* \cdot A^{-1} \beta_1 = \mathcal{T}_1^* \cdot \beta_1 \quad \text{where} \quad \mathcal{T}_1 = A^{-*} \mathcal{T}$$

- So the generalized velocities and forces go together – compatible pairs defined by the power relationship
- Transforming the generalized velocities, transforms the generalized forces as well
- Examples
 - Say doubling
 - Picking different units
 - Picking combinations – relative to absolute angle rates

Multibody state and state derivatives



- State: $\mathbf{x} = (\theta, \beta)$
- State derivative: $\dot{\mathbf{x}} = (\dot{\theta}, \dot{\beta})$
- Next look at equations of motion for single rigid body

$$\dot{\mathbf{x}}(t) = \mathbf{g}(\mathbf{x}(t), \mathbf{u}(t))$$

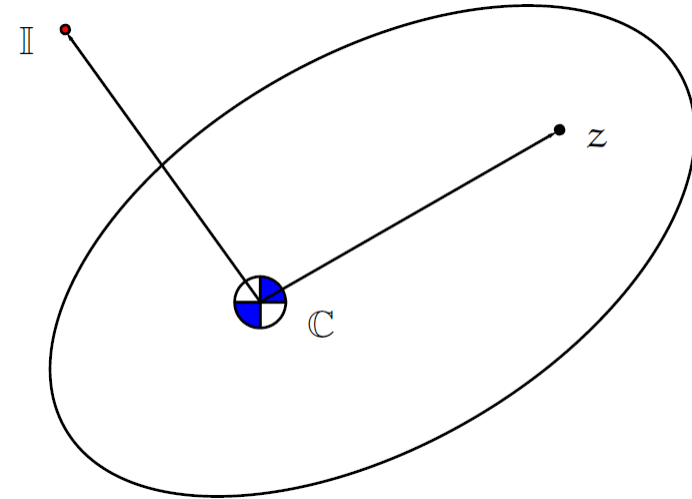


Single Rigid Body



Equations of motion

- More than the generalized coordinates, the choice of generalized velocities directly effects the form of the equations of motion
- We now look at the equations of motion of a single rigid body for different choices of generalized velocities



Single rigid body generalized velocities



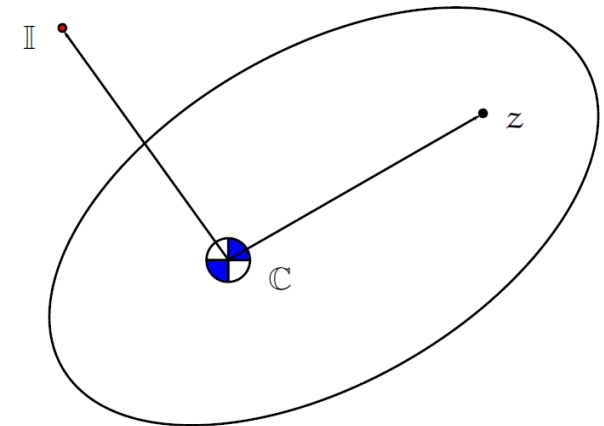
Generalized velocity coordinate options

$$\beta_{\mathcal{J}} = {}^{\mathbb{I}}\mathcal{V}(\mathbb{C}) \quad \text{sp. velocity of CM, inertial frame}$$

$$\beta_{\mathcal{J}} = {}^{\mathbb{I}}\mathcal{V}(\mathbf{z}) \quad \text{sp. velocity of } \mathbf{z}, \text{ inertial frame}$$

$$\beta_{\mathbb{B}} = {}^{\mathbb{B}}\mathcal{V}(\mathbf{z}) \quad \text{sp. velocity of } \mathbf{z}, \text{ body frame}$$

$$\beta_{\mathbb{I}} \triangleq {}^{\mathbb{I}}\mathcal{V}_{\mathbb{I}} \triangleq \phi^*(\mathbb{C}, \mathbb{I})\mathcal{V}(\mathbb{C}) \quad \text{inertial reference point } \mathbb{I} \text{ sp. vel}$$



All of these options include angular velocity coordinates, and are hence quasi-velocities.



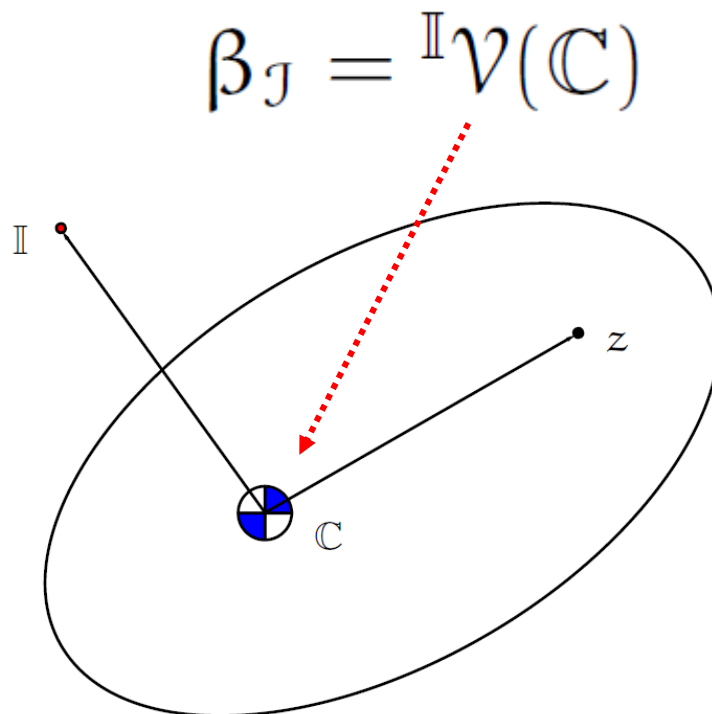
Single rigid body

Center of Mass Dynamics

Generalized velocities – center of mass, inertial derivative



Coordinates of the spatial velocity of the center of mass (CM) in inertial frame



Center of mass dynamics



- Derivative of linear and angular momenta at 3D component level

$$\overset{\text{angular}}{N(\mathbb{C})} = \frac{d_{\mathbb{I}} \mathcal{J}(\mathbb{C}) \omega(\mathbb{C})}{dt} \quad \text{and} \quad \overset{\text{linear}}{F(\mathbb{C})} = \frac{d_{\mathbb{I}} m v(\mathbb{C})}{dt}$$

- Equivalently using spatial notation

$$\overset{\text{spatial}}{f(\mathbb{C})} = \frac{d_{\mathbb{I}} M(\mathbb{C}) \mathcal{V}(\mathbb{C})}{dt} = \frac{d_{\mathbb{I}} \mathbf{h}(\mathbb{C})}{dt}$$

- Spatial momentum is conserved in the absence of external spatial forces

Some notation



Component forms of spatial vectors

$$X = X^\omega + X^v$$

decomposition

$$\mathcal{V}(x) \triangleq \begin{bmatrix} \omega(x) \\ v(x) \end{bmatrix}$$

$\mathcal{V}^\omega(x) \triangleq \begin{bmatrix} \omega(x) \\ \mathbf{0} \end{bmatrix}$

angular component

$\mathcal{V}^v(x) \triangleq \begin{bmatrix} \mathbf{0} \\ v(x) \end{bmatrix}$

linear component

$$\tilde{\mathcal{V}}^\omega(z) = \begin{pmatrix} \tilde{\omega} & \mathbf{0} \\ \mathbf{0} & \tilde{\omega} \end{pmatrix}$$

Time derivative of spatial inertia



- Inertial and body frame spatial inertia relationship

$${}^{\mathbb{I}}\mathcal{M}(z) = \begin{pmatrix} {}^{\mathbb{I}}\mathcal{R}_{\mathbb{B}} & \mathbf{0} \\ \mathbf{0} & {}^{\mathbb{I}}\mathcal{R}_{\mathbb{B}} \end{pmatrix} {}^{\mathbb{B}}\mathcal{M}(z) \begin{pmatrix} {}^{\mathbb{B}}\mathcal{R}_{\mathbb{I}} & \mathbf{0} \\ \mathbf{0} & {}^{\mathbb{B}}\mathcal{R}_{\mathbb{I}} \end{pmatrix}$$

- Time derivative of the inertial spatial inertia

$$\begin{aligned} \dot{{}^{\mathbb{I}}\mathcal{M}}(z) &= \begin{pmatrix} \tilde{\omega} & \mathbf{0} \\ \mathbf{0} & \tilde{\omega} \end{pmatrix} {}^{\mathbb{B}}\mathcal{M}(z) - \mathcal{M}(z) \begin{pmatrix} \tilde{\omega} & \mathbf{0} \\ \mathbf{0} & \tilde{\omega} \end{pmatrix} \\ &= \underline{\tilde{\mathcal{V}}^{\omega}(z)} \mathcal{M}(z) - \mathcal{M}(z) \underline{\tilde{\mathcal{V}}^{\omega}(z)} \end{aligned}$$



CM equations of motion

The CM equations of motion are

$$\underset{\substack{\text{sp. force at} \\ \text{CM}}}{\mathbf{f}(\mathbb{C})} = \underset{\substack{\text{sp. inertia at} \\ \text{CM}}}{\mathbf{M}(\mathbb{C})} \underset{\text{gen. accel}}{\dot{\boldsymbol{\beta}}_{\mathcal{J}}(\mathbb{C})} + \underset{\text{gyroscopic}}{\mathbf{b}_{\mathcal{J}}(\mathbb{C})}$$

with gyroscopic term

$$\begin{aligned} \mathbf{b}_{\mathcal{J}}(\mathbb{C}) &\triangleq \tilde{\mathbf{v}}^{\omega}(\mathbb{C})\mathbf{M}(\mathbb{C})\mathbf{v}^{\omega}(\mathbb{C}) \\ &= \bar{\mathbf{v}}^{\omega}(\mathbb{C})\mathbf{M}(\mathbb{C})\mathbf{v}^{\omega}(\mathbb{C}) = \begin{bmatrix} \tilde{\omega} \mathcal{J}(\mathbb{C}) \omega \\ \mathbf{0} \end{bmatrix} \end{aligned}$$

familiar component level term
↓

CM equations of motion (derivation)



Have $f(\mathbb{C}) = \frac{d_{\mathbb{I}}M(\mathbb{C})\mathcal{V}(\mathbb{C})}{dt}$ ← *spatial momentum*

$$f(\mathbb{C}) \stackrel{2.21}{=} M(\mathbb{C})\dot{\beta}_{\mathbb{J}}(\mathbb{C}) + \frac{d_{\mathbb{I}}M(\mathbb{C})}{dt}\mathcal{V}(\mathbb{C})$$

$$\stackrel{2.22}{=} M(\mathbb{C})\dot{\beta}_{\mathbb{J}}(\mathbb{C}) + \underbrace{\left[\tilde{\mathcal{V}}^{\omega}(\mathbb{C})M(\mathbb{C}) - M(\mathbb{C})\tilde{\mathcal{V}}^{\omega}(\mathbb{C}) \right]}_{\dot{M}} \mathcal{V}(\mathbb{C})$$

$$\stackrel{1.25}{=} M(\mathbb{C})\dot{\beta}_{\mathbb{J}}(\mathbb{C}) + \tilde{\mathcal{V}}^{\omega}(\mathbb{C})M(\mathbb{C})\mathcal{V}^{\omega}(\mathbb{C})$$

$$+ \left[\tilde{\mathcal{V}}^{\omega}(\mathbb{C})M(\mathbb{C})\mathcal{V}^{\nu}(\mathbb{C}) - M(\mathbb{C})\tilde{\mathcal{V}}^{\omega}(\mathbb{C})\mathcal{V}^{\nu}(\mathbb{C}) \right]$$

$$\stackrel{2.10}{=} M(\mathbb{C})\dot{\beta}_{\mathbb{J}}(\mathbb{C}) + \tilde{\mathcal{V}}^{\omega}(\mathbb{C})M(\mathbb{C})\mathcal{V}^{\omega}(\mathbb{C})$$

CM gyroscopic term



$$\begin{aligned} \mathbf{b}_j(\mathbb{C}) &\triangleq \tilde{\mathbf{v}}^\omega(\mathbb{C})\mathbf{M}(\mathbb{C})\mathcal{V}^\omega(\mathbb{C}) \\ &= \bar{\mathbf{v}}^\omega(\mathbb{C})\mathbf{M}(\mathbb{C})\mathcal{V}^\omega(\mathbb{C}) = \begin{bmatrix} \tilde{\omega} \mathcal{J}(\mathbb{C}) \omega \\ \mathbf{0} \end{bmatrix} \end{aligned}$$

The gyroscopic term in the CM equations of motion does no work, i.e.

$$\mathcal{V}^*(\mathbb{C})\mathbf{b}_j(\mathbb{C}) = \mathbf{0}$$



Single rigid body

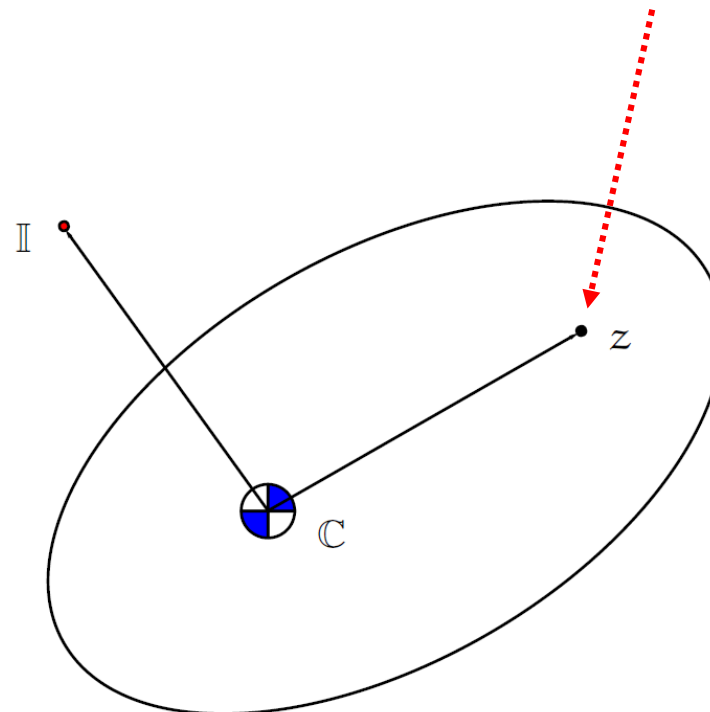
General point dynamics
Inertial derivatives

Generalized velocities – arbitrary point, inertial derivative



Coordinates of spatial velocity of z in inertial frame

$$\beta_g = {}^{\text{II}}\mathcal{V}(z)$$





Generalized velocities – arbitrary point, inertial derivative

- Generalized velocity – spatial velocity in inertial frame

$$\beta_J = {}^{\mathbb{I}}\mathcal{V}(z)$$

- General acceleration relationship for $\mathcal{V}(y) = \phi^*(x, y) \mathcal{V}(x)$

$$\begin{aligned} \dot{\beta}_J(y) &\triangleq \frac{d_{\mathbb{I}}\mathcal{V}(y)}{dt} = \phi^*(x, y) \dot{\beta}_J(x) - \tilde{\mathcal{V}}(y) \mathcal{V}(x) \\ &= \phi^*(x, y) \dot{\beta}_J(x) + \begin{bmatrix} \mathbf{0} \\ \tilde{\omega} \tilde{\omega} l(x, y) \end{bmatrix} \leftarrow \text{Coriolis term} \end{aligned}$$

- Relationship to CM generalized velocities

$$\dot{\beta}_J(z) \triangleq \frac{d_{\mathbb{I}}\mathcal{V}(z)}{dt} = \phi^*(\mathbb{C}, z) \dot{\beta}_J(\mathbb{C}) + \begin{bmatrix} \mathbf{0} \\ -\tilde{\omega} \tilde{\omega} p(z) \end{bmatrix}$$

Equations of Motion with $\beta_j = {}^{\mathbb{I}}\mathcal{V}(z)$



The equations of motion with coordinates of generalized velocities of point z in inertial frame

$$\mathbf{f}(z) = \mathbf{M}(z)\dot{\beta}_j(z) + \mathbf{b}_j(z)$$

with gyroscopic term

$$\begin{aligned}\mathbf{b}_j(z) &\triangleq \tilde{\mathbf{V}}^\omega(z)\mathbf{M}(z)\mathcal{V}^\omega(z) = \bar{\mathbf{V}}^\omega(z)\mathbf{M}(z)\mathcal{V}^\omega(z) \\ &= \begin{bmatrix} \tilde{\omega} \mathcal{J}(z)\omega \\ m \tilde{\omega} \tilde{\omega} p(z) \end{bmatrix}\end{aligned}$$

Derivation of equations of motion



Uses CM equations of motion

$$\begin{aligned}
 M(z)\dot{\beta}_J(z) &\stackrel{2.25}{=} \underbrace{M(z)} \left(\underbrace{\phi^*(\mathbb{C}, z)} \dot{\beta}_J(\mathbb{C}) + \begin{bmatrix} \mathbf{0} \\ -\tilde{\omega} \tilde{\omega} p(z) \end{bmatrix} \right) \\
 &\stackrel{2.12, 2.7}{=} \phi(z, \mathbb{C}) M(\mathbb{C}) \dot{\beta}_J(\mathbb{C}) - \begin{bmatrix} m \tilde{p}(z) \tilde{\omega} \tilde{\omega} p(z) \\ m \tilde{\omega} \tilde{\omega} p(z) \end{bmatrix} \\
 &\stackrel{2.23, A.1}{=} \underbrace{\phi(z, \mathbb{C})} \left(\underbrace{f(\mathbb{C})} - \begin{bmatrix} \tilde{\omega} \mathcal{J}(\mathbb{C}) \omega \\ \mathbf{0} \end{bmatrix} \right) - \begin{bmatrix} -m \tilde{\omega} \tilde{p}(z) \tilde{p}(z) \omega \\ m \tilde{\omega} \tilde{\omega} p(z) \end{bmatrix} \\
 &\stackrel{1.66}{=} f(z) - \begin{bmatrix} \tilde{\omega} (\mathcal{J}(\mathbb{C}) - m \tilde{p}(z) \tilde{p}(z)) \omega \\ m \tilde{\omega} \tilde{\omega} p(z) \end{bmatrix} \\
 &\stackrel{2.11}{=} f(z) - \begin{bmatrix} \tilde{\omega} \mathcal{J}(z) \omega \\ m \tilde{\omega} \tilde{\omega} p(z) \end{bmatrix}
 \end{aligned}$$

Gyroscopic term with $\beta_{\mathcal{J}} = {}^{\mathbb{I}}\mathcal{V}(z)$



- Unlike at CM, the gyroscopic forces do work
- Moreover, the spatial momentum about z is not constant in the inertial frame in the absence of external forces!

$$f(z) \neq \frac{d_{\mathbb{I}}h(z)}{dt}$$



Single rigid body

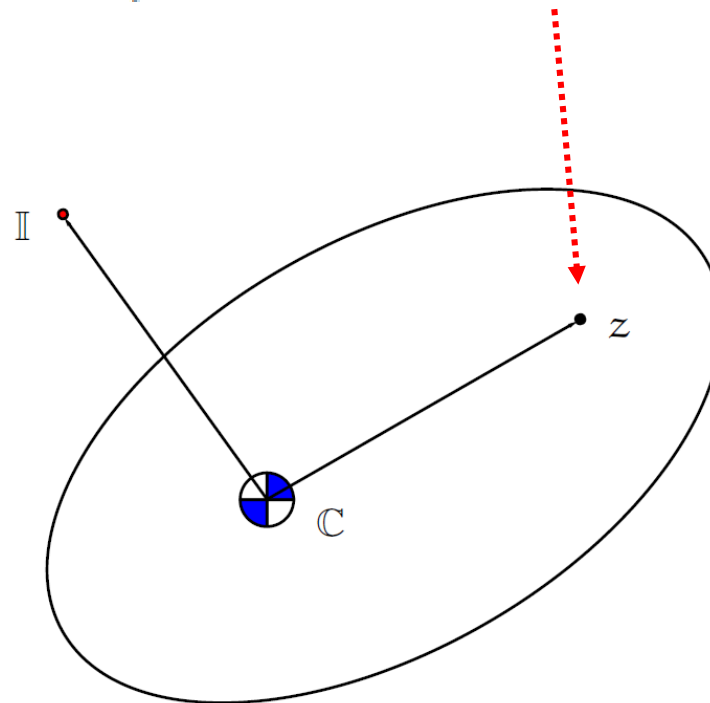
General point dynamics
Body derivatives

Generalized velocities – arbitrary point, body derivative



Coordinates of spatial velocity of z in body frame

$$\beta_{\mathbb{B}} = {}^{\mathbb{B}}\mathcal{V}(z)$$



Generalized velocity relationship



Relationship between body frame and inertial frame generalized velocities

$$\beta_{\mathbb{B}} = {}^{\mathbb{B}}\mathcal{V}(z) \quad \beta_{\mathbb{J}} = {}^{\mathbb{I}}\mathcal{V}(z)$$

$$\dot{\beta}_{\mathbb{B}}(z) = \dot{\beta}_{\mathbb{J}}(z) - \begin{bmatrix} 0 \\ \tilde{\omega} v(z) \end{bmatrix}$$

Equations of Motion with $\beta_{\mathbb{B}} = {}^{\mathbb{B}}\mathcal{V}(z)$



The equations of motion with coordinates of generalized velocities of point z in body frame

$$\mathbf{f}(z) = \mathbf{M}(z)\dot{\beta}_{\mathbb{B}}(z) + \mathbf{b}(z)$$

with gyroscopic term

$$\mathbf{b}(z) \triangleq \bar{\mathbf{V}}(z)\mathbf{h}(z) = \bar{\mathbf{V}}(z)\mathbf{M}(z)\mathcal{V}(z)$$

Derivation of equations of motion



$$\begin{aligned}
 M(z)\dot{\beta}_{\mathbb{B}}(z) &\stackrel{2.19}{=} M(z) \left(\dot{\beta}_{\mathcal{J}}(z) - \begin{bmatrix} 0 \\ \tilde{\omega} v(z) \end{bmatrix} \right) \\
 &\stackrel{2.26,2.7}{=} f(z) - \begin{bmatrix} \tilde{\omega} \mathcal{J}(z)\omega \\ m \tilde{\omega} \tilde{\omega} p(z) \end{bmatrix} - \begin{bmatrix} m \tilde{p}(z) \tilde{\omega} v(z) \\ m \tilde{\omega} v(z) \end{bmatrix} \\
 &= f(z) - \begin{bmatrix} \tilde{\omega} \mathcal{J}(z)\omega + m \tilde{p}(z) \tilde{\omega} v(z) \\ m \tilde{\omega} \tilde{\omega} p(z) + m \tilde{\omega} v(z) \end{bmatrix} \\
 &\stackrel{A.1}{=} f(z) - \begin{bmatrix} \tilde{\omega} \mathcal{J}(z)\omega - m(\tilde{\omega} \tilde{v}(z)p(z) + \tilde{v}(z) \tilde{p}(z)\omega) \\ m \tilde{\omega}(\tilde{\omega} p(z) + v(z)) \end{bmatrix} \\
 &= f(z) - \begin{bmatrix} \tilde{\omega}(\mathcal{J}(z)\omega + m \tilde{p}(z)v(z)) - m \tilde{v}(z) \tilde{p}(z)\omega \\ m \tilde{\omega}(\tilde{\omega} p(z) + v(z)) \end{bmatrix} \\
 &= f(z) - \bar{V}(z) \begin{bmatrix} \mathcal{J}(z)\omega + m \tilde{p}(z)v(z) \\ -m \tilde{p}(z)\omega + mv(z) \end{bmatrix} \\
 &\stackrel{2.7}{=} f(z) - \bar{V}(z)M(z)V(z) \stackrel{2.16}{=} f(z) - \bar{V}(z)h(z)
 \end{aligned}$$



Gyroscopic term with $\beta_{\mathbb{B}} = \mathbb{B}\mathcal{V}(z)$

- Once again, the gyroscopic spatial force does no work

$$\mathcal{V}^*(z)\mathbf{b}(z) \stackrel{2.28}{=} \mathcal{V}^*(z)\bar{\mathcal{V}}(z)\mathbf{h}(z) \stackrel{1.27}{=} 0$$

- Kinetic energy conservation easy to verify

$$\frac{1}{2} \frac{d\mathcal{V}^*(z)M(z)\mathcal{V}(z)}{dt} = \mathcal{V}^*(z)M(z)\dot{\beta}_{\mathbb{B}}(z) \stackrel{2.28}{=} -\mathcal{V}^*(z)\mathbf{b}(z) \stackrel{2.29}{=} 0$$

- Spatial momentum about z not conserved



Single rigid body

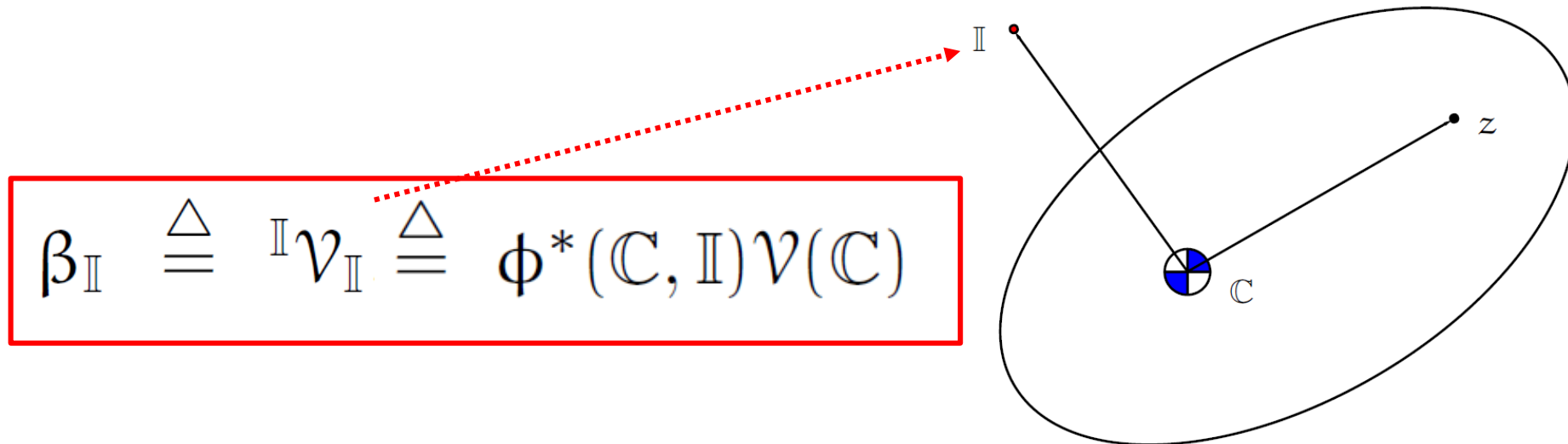
Inertial Reference Point dynamics

Generalized velocities – inertially referenced spatial velocity



Coordinates of inertially referenced spatial velocity

- Spatial velocity of \mathbb{I} , as if the frame were rigidly attached to the body



Inertially referenced spatial velocity



The inertially referenced spatial velocity is the same for all points x on the rigid body!

$$\mathcal{V}_{\mathbb{I}} = \Phi^*(x, \mathbb{I}) \mathcal{V}(x)$$

*does not depend
on the choice of x*

Inertially referenced spatial Inertia



Inertially referenced spatial inertia

$$M_{\mathbb{I}} \triangleq \phi(\mathbb{I}, \mathbb{C}) M(\mathbb{C}) \phi^*(\mathbb{I}, \mathbb{C}) = \begin{pmatrix} \mathcal{J}_{\mathbb{I}} & m \tilde{p}_{\mathbb{I}} \\ -m \tilde{p}_{\mathbb{I}} & m \mathbf{I}_3 \end{pmatrix}$$

parallel axis theorem

and its time derivative

$$\dot{M}_{\mathbb{I}} \triangleq \frac{d_{\mathbb{I}} M_{\mathbb{I}}}{dt} = \bar{v}_{\mathbb{I}} M_{\mathbb{I}} - M_{\mathbb{I}} \tilde{v}_{\mathbb{I}} = \bar{v}_{\mathbb{I}} M_{\mathbb{I}} + M_{\mathbb{I}} (\bar{v}_{\mathbb{I}})^*$$

Equations of motion with $\beta_{\text{II}} \triangleq {}^{\text{II}}\mathcal{V}_{\text{II}}$



The equations of motion with coordinates of the inertially referenced spatial velocity

$$f_{\text{II}} = M_{\text{II}} \dot{\beta}_{\text{II}} + b_{\text{II}}$$

with gyroscopic term

$$b_{\text{II}} \triangleq \dot{M}_{\text{II}} \mathcal{V}_{\text{II}} = \bar{\mathcal{V}}_{\text{II}} M_{\text{II}} \mathcal{V}_{\text{II}}$$

Gyroscopic term with $\beta_{\text{II}} \triangleq {}^{\text{II}}\mathcal{V}_{\text{II}}$



- Once again, the gyroscopic spatial force does no work
- Kinetic energy conservation easy to verify
- Inertially referenced spatial momentum is conserved.

Summary of equations of motion



The following summarizes the properties of the equations of motion from the different choice for generalized velocities

Formulation	\mathbb{I} deriv, \mathbb{C}	\mathbb{I} deriv, z	\mathbb{B} deriv, z	Inertial ref, \mathbb{I}
Section	2.3.1	2.3.2	2.4	2.5
Gen. vel. β	${}^{\mathbb{I}}\mathcal{V}(\mathbb{C})$	${}^{\mathbb{I}}\mathcal{V}(z)$	${}^{\mathbb{B}}\mathcal{V}(z)$	$\mathcal{V}_{\mathbb{I}}$
Gyroscopic force \mathfrak{b}	$\bar{\mathcal{V}}^{\omega}(\mathbb{C})\mathcal{M}(\mathbb{C})\mathcal{V}^{\omega}(\mathbb{C})$	$\bar{\mathcal{V}}^{\omega}(z)\mathcal{M}(z)\mathcal{V}^{\omega}(z)$	$\bar{\mathcal{V}}(z)\mathfrak{h}(z)$	$\bar{\mathcal{V}}_{\mathbb{I}}\mathfrak{h}_{\mathbb{I}}$
Conserved spatial momentum	✓			✓
Non-working \mathfrak{b}	✓		✓	✓
Independent of linear velocity	✓	✓		

Summary



- Looked at defining the dynamical system for multibody systems
- Looked at the choice of generalized coordinates, velocities and forces
- Developed equations of motion of a single rigid body using spatial notation
- Examined the impact of changing the generalized velocities on the equations of motion

SOA Foundations Track Topics (serial-chain rigid body systems)



1. **Spatial (6D) notation** – spatial velocities, forces, inertias; spatial cross-product, rigid body transformations & properties; parallel axis theorem
2. **Single rigid body dynamics** – equations of motion about arbitrary frame using spatial notation
3. **Serial-chain kinematics** – minimal coordinate formulation, hinges, velocity recursions, Jacobians; first spatial operators; $O(N)$ scatter and gather recursions
4. **Serial-chain dynamics** – equations of motion using spatial operators; Newton–Euler mass matrix factorization; $O(N)$ inverse dynamics; composite rigid body inertia; forward Lyapunov equation; mass matrix decomposition; mass matrix computation; alternative inverse dynamics
5. **Articulated body inertia** - Concept and definition; Riccati equation; alternative force decompositions
6. **Mass matrix factorization and inversion** – spatial operator identities; Innovations factorization of the mass matrix; Inversion of the mass matrix
7. **Recursive forward dynamics** – $O(N)$ recursive forward dynamics algorithm; including gravity and external forces; inter-body forces identity